

Functional Description

The valve consists of a seat by-pass, relief valve fitted The ratio of control designates a ratio of surfaces with an auxiliary control with a differential piston and of the differential slide valve cross-section area and its by-pass single-way valve serving for reverse direction seat. Therefore, the necessary control pressure of flow. The liquid is flowing through the single-way valve for opening the valve does not correspond to from the channel (2) to the channel (1) with a small pressure drop. In the opposite direction the single-way valve on the rear side of which a gate valve seat is fitted is pressed through the action of a spring and the load pressure against the spring-loaded valve gate valve. In this way the valve is nearly closed hermetically. If the pressure in the channel (1) exceeds a set up value of the spring force the gate valve is pressed out of the seat and the overpressure in that case is relieved into channel (2). For ensuring the function of holding the load the spring force should be set up to a value by 30 % higher when compared to an expected pressure exerted by the load.

If the load has to be moved it is possible to ensure it with the help of so called auxiliary control from the channel (3) by introducing already certain control pressure.

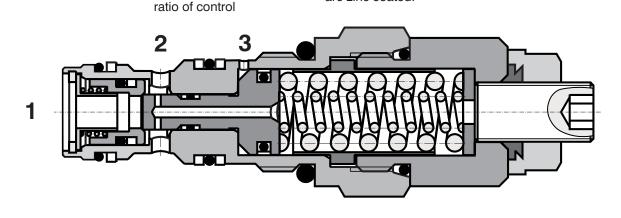
The control pressure is calculated in the following way:

Control pressure = set up pressure - load pressure

the difference between the set up pressure and load pressure however; it corresponds to the ratio of this difference and the control ratio. In the formula as mentioned above it is necessary to take into consideration that in differential cylinders it is necessary to add to the control ratio also the appropriate ratio of piston surfaces in the direction of movement.

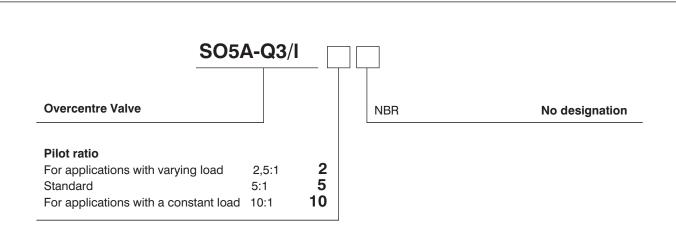
As soon as the control pressure attains a necessary value the differential gate valve is moved out from the seat and then the way from the channel (1) to the channel (2) is released. If now the load tries to accelerate and be fast as for the oil supply the supply pressure decreases, therefore, also the control pressure in the channel (3) is decreased. The spring force tries to shut off the valve again, therefore, in consequence of which the flow from the consumer decreases and the inlet pressure to the consumer increases again. In this way it is ensured aconstant inlet pressure by means of which the movement of the load can be controlled.

As for appropriate basic surface finish the external parts are zinc coated.



HA 5200

Ordering Code

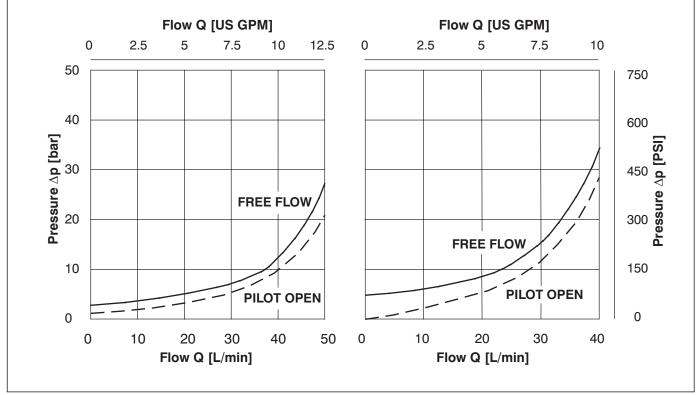


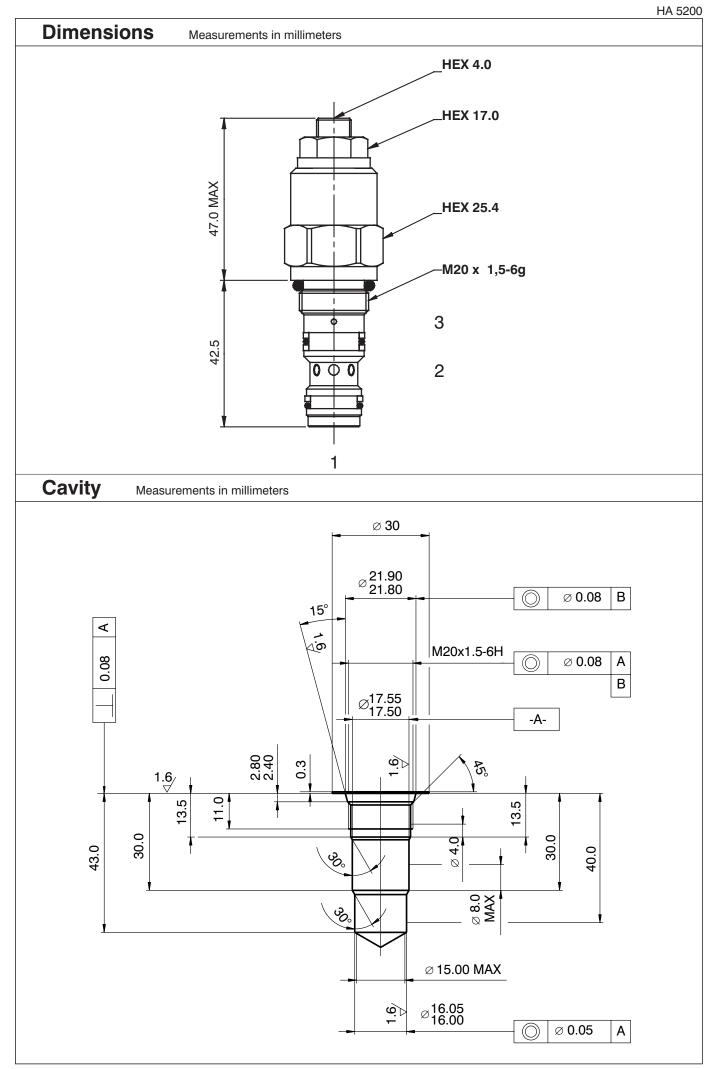
Technical Data

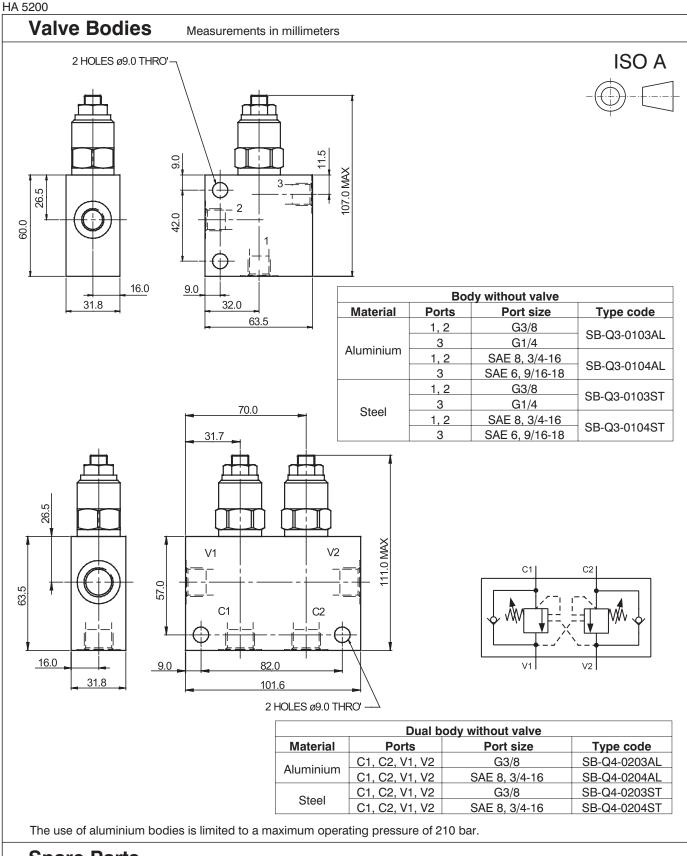
	M20 x 1,5
L/min	30
bar	270
bar	350
bar	see ∆p - Q characteristics
	Hydraulic oil (HL, HLP) according to DIN 51524
°C	-20 +90
mm ² /s	20400
	according to ISO 4406 (1999), Class 21/18/15
kg	0,15
Nm	45 ⁺²
	any
	bar bar bar °C mm ² /s kg

∆p-Q Characteristics

Measured at $v = 40 \text{ mm}^2/\text{s}$







Spare Parts

Seal kits on request.

Caution!

- The packing foil is recyclable.
- The technical information regarding the product presented in this catalogue is for descriptive purposes only. It should not be construed in any case as a guaranteed representation of the product properties in the sense of the law.

ARGO-HYTOS s.r.o. CZ - 543 15 Vrchlabí Tel.: +420-499-403111, Fax: +420-499-403421 E-mail: sales.cz@argo-hytos.com www.argo-hytos.com